

# Femtet Seminar

Understanding

# Piezoelectric Analysis &

Acoustic Analysis

202009

### Table of Contents



### **☆**Piezoelectric Analysis

- 1. Case Studies
- 2. Functions
- 3. Points

### **☆**Acoustic Analysis

- 4. Case Studies
- 5. Functions
- 6. Points

### **☆**Piezoelectric-Acoustic Coupled Analysis

- 7. Case Studies
- 8. Points

### Table of Contents



### **☆**Piezoelectric Analysis

- 1. Case Studies
- 2. Functions
- 3. Points

### **☆**Acoustic Analysis

- 4. Case Studies
- 5. Functions
- 6. Points

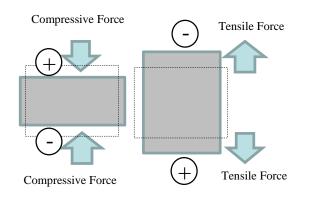
### ☆Piezoelectric-Acoustic Coupled Analysis

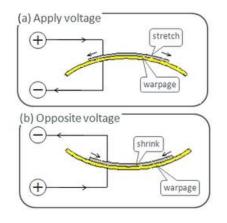
- 7. Case Studies
- 8. Points

#### Piezoelectric Analysis

### Case Studies

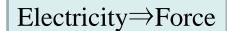




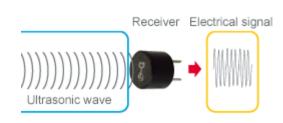




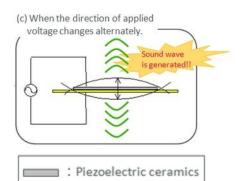
### Force⇒Electricity



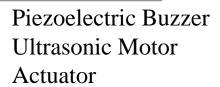
### Electricity ⇒ Force ⇒ Electricity



Ultrasonic Sensor Acceleration Sensor Piezoelectric Gyro Shock Sensor



: Metal plate





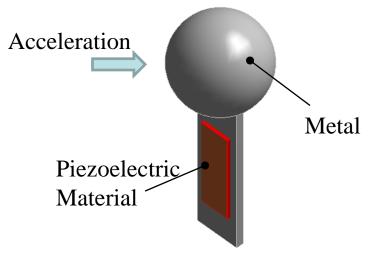
Crystal

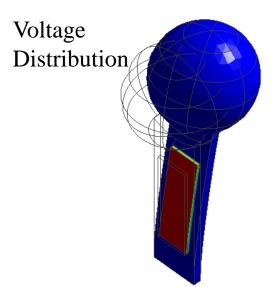
SAW Filter Crystal Oscillator

# Force to Electricity

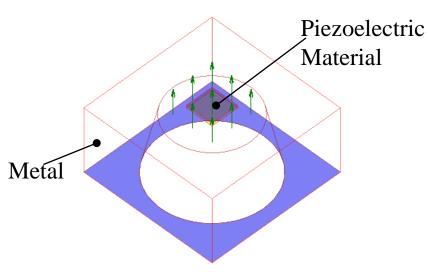


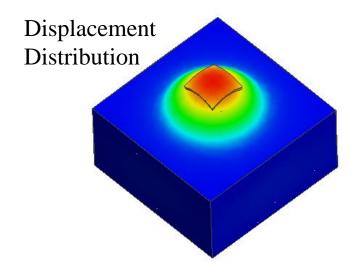
### Acceleration Sensor





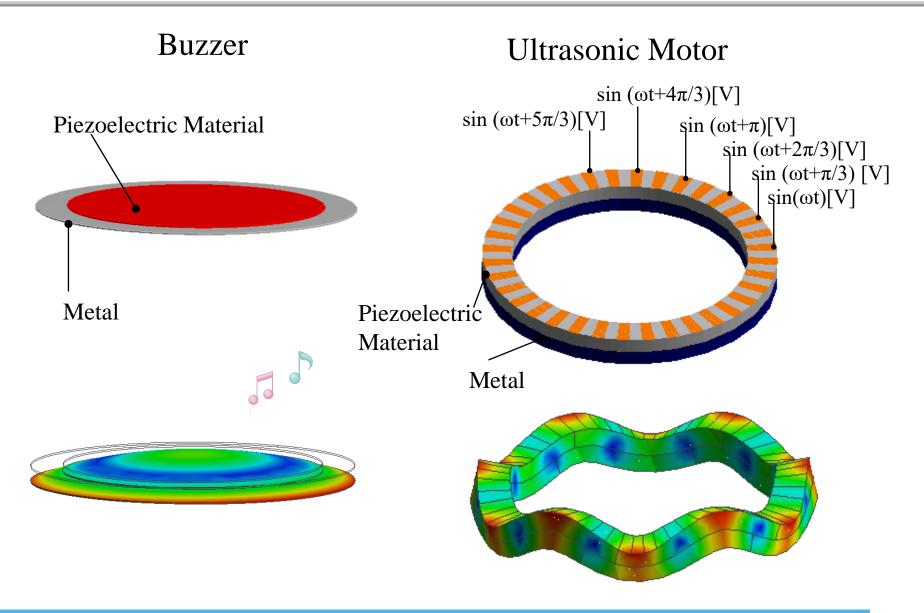
### Pressure Sensor





# Electricity to Force

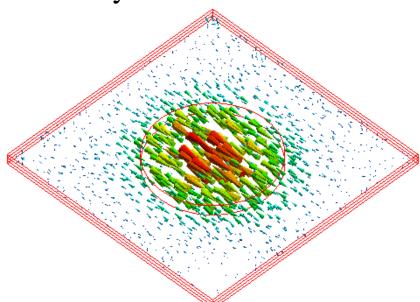




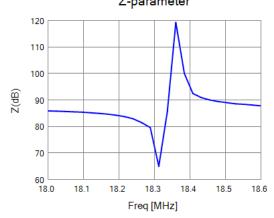
# Electricity to Force to Electricity



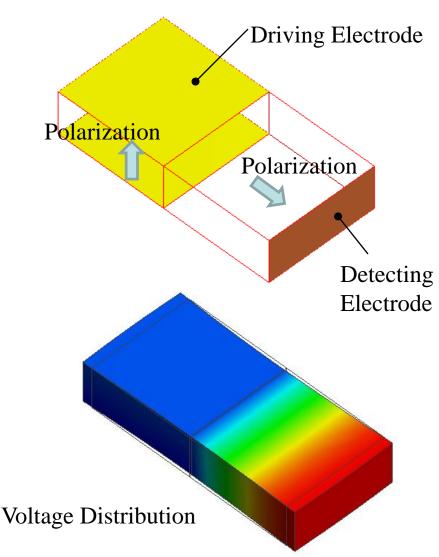
### Crystal Oscillator



### Impedance Characteristics Z-parameter



### Piezoelectric Transformer

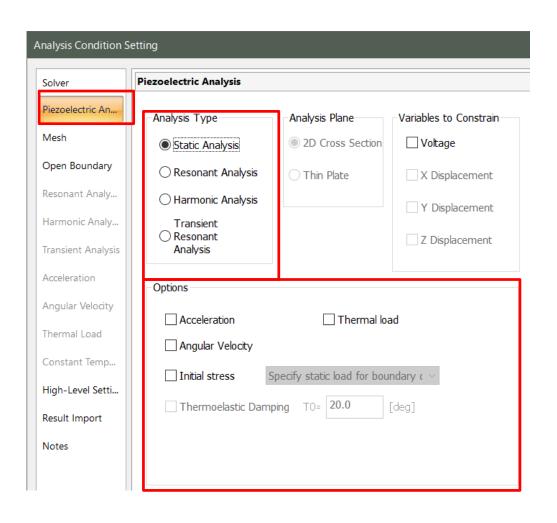


# 2. Functions of Piezoelectric Analysis Murata Software

- (1) Analysis Condition
- (2) Boundary Condition
- (3) Material Property
- (4) Body Attribute
- (5) Results Display

# (1) Analysis Condition





#### Analysis Type

- •Static
- Harmonic
- Resonant
- Transient

#### **Options**

- Acceleration
- Thermal Load
- Angular Velocity
- Initial Stress Taken into Account

#### Analysis Plane

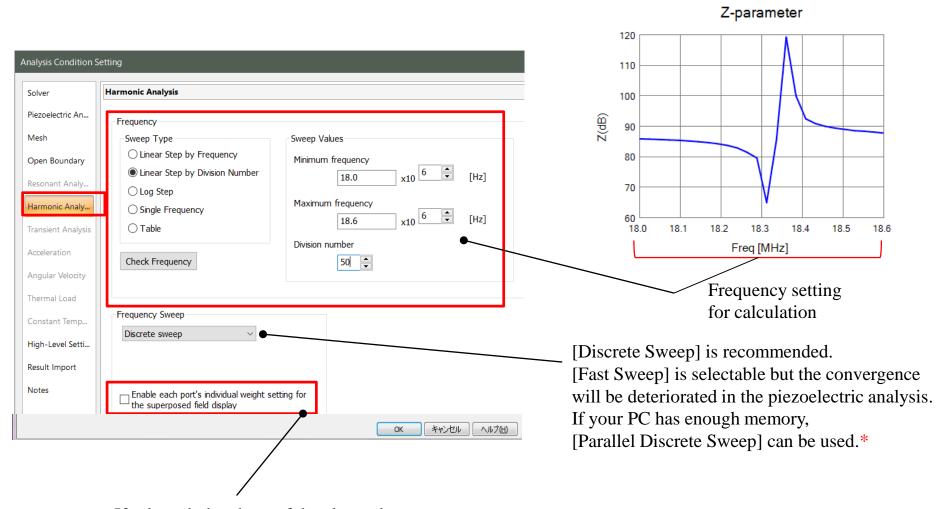
Used for 2D analysis

#### Variables to Constrain

Select [Voltage] to couple with acoustic analysis without taking piezoelectricity into account

# Harmonic Analysis

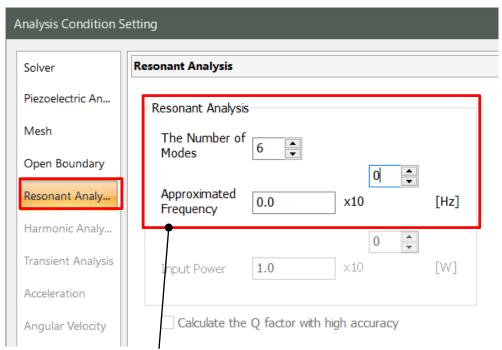




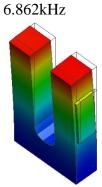
If selected, the phase of the electrode can be changed after calculation.

<sup>\*</sup> Option for Accelerator is required.

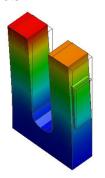
### Resonant Analysis



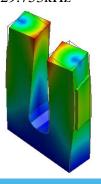
[The Number of Modes]: Calculations done by this number [Approximated Frequency]: Calculates frequencies near to this value



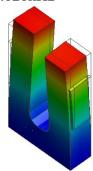
8.322kHz



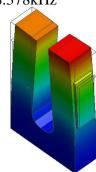
29.753kHz



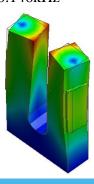
7.020kHz



8.378kHz

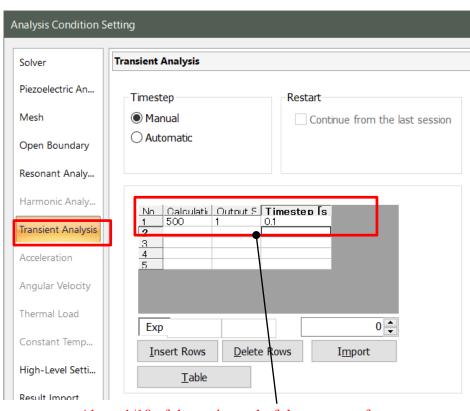


30.146kHz



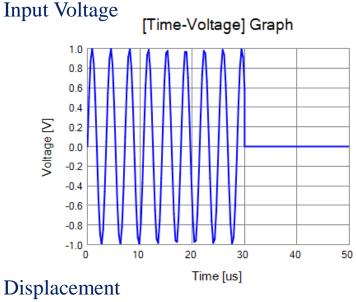
### Transient Analysis



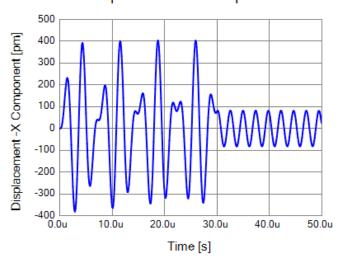


About 1/10 of the reciprocal of the resonant frequency

The modes obtained in the resonant analysis are used. [Inverse Fourier transform] is applied.



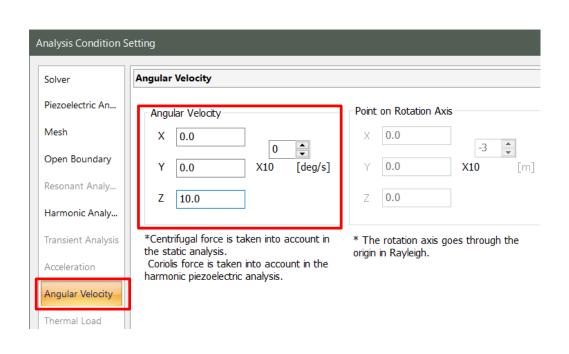
#### Displacement -X Component

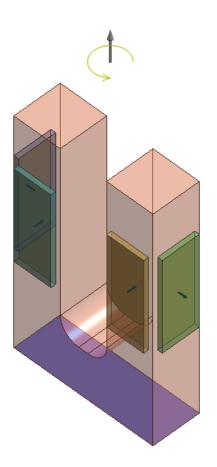


# Angular Velocity



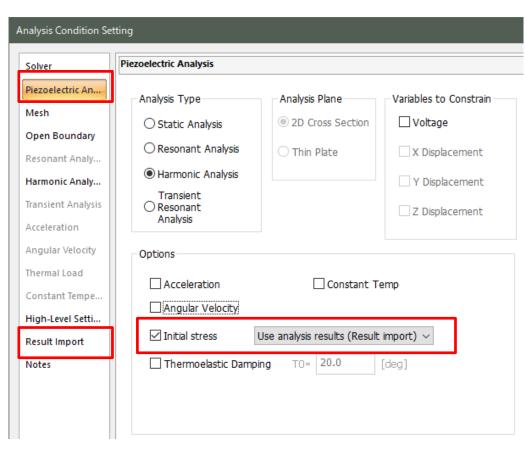
By specifying the angular velocity, Coriolis force can be taken into account.



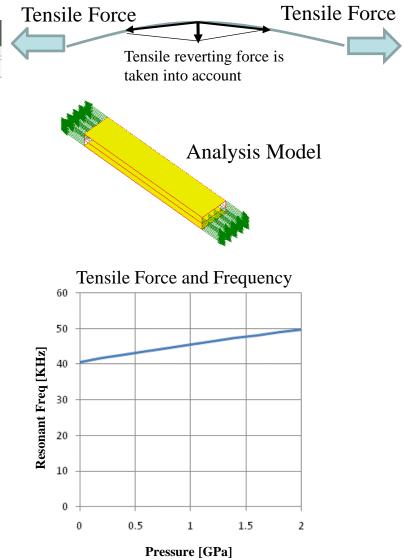


### Analysis with Initial Stress Taken into Account





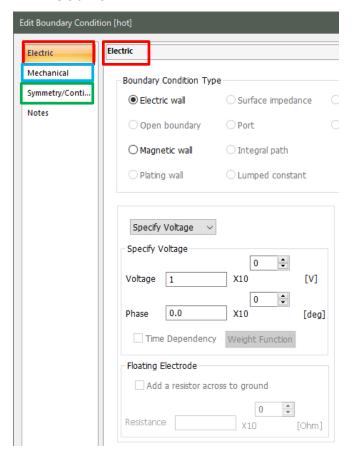
At first, static analysis is executed. Its results are imported by [Result Import]. Harmonic and resonant analysis are executed.



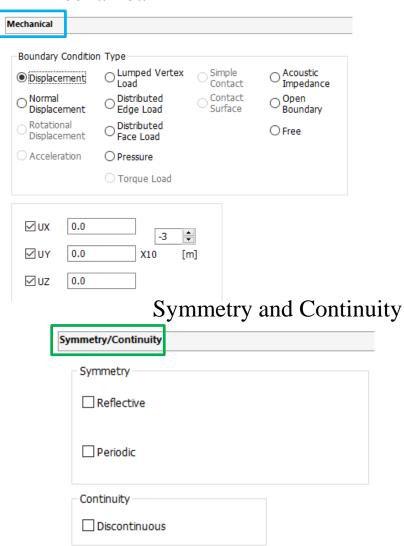
# (2) Boundary Condition



#### Electric



#### Mechanical



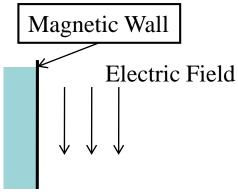
# Electric Boundary Condition



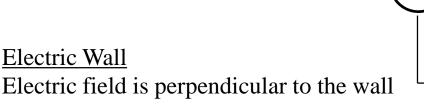
#### Magnetic Wall

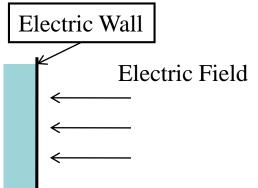
Electric Wall

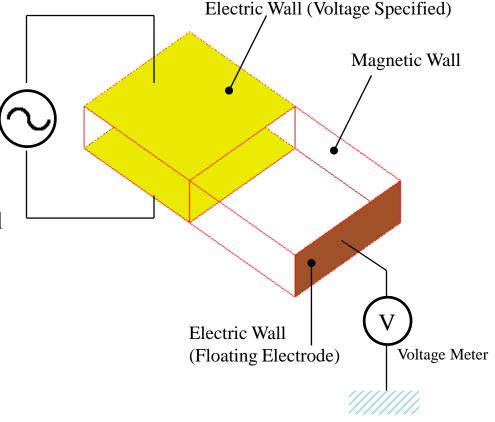
Electric field is parallel to the wall



Example: Piezoelectric Transformer

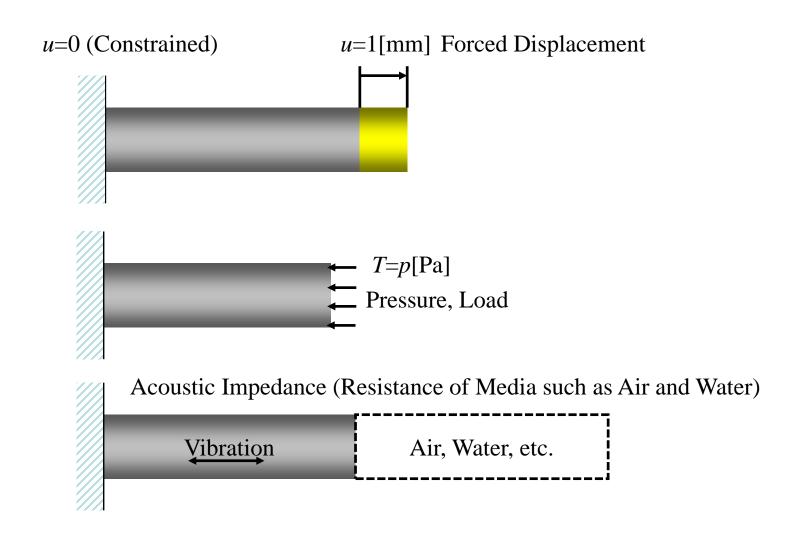






# Mechanical Boundary Condition Murata Software





# Example: Acoustic Impedance



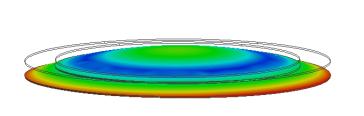
Acoustic impedance represents the resistance of media such as air or water. By applying the acoustic impedance as a boundary condition to the face that is in contact with the media, the resistance of air or water can be taken into account.

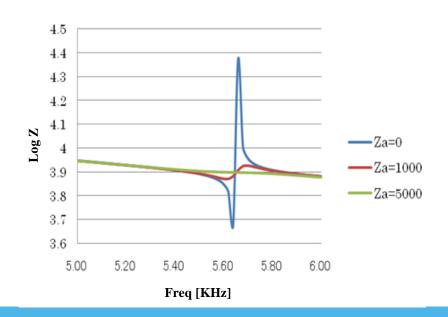
In the harmonic analysis, acoustic impedance is expressed by Z= $\rho$ c where  $\rho$  is density, c is sound speed of medium.

#### [Examples]

Air:  $Z = 1.205[kg/m3]*340[m/s]=409.7[kg/(m2 \cdot s)]=409.7[N \cdot s/m3]$ 

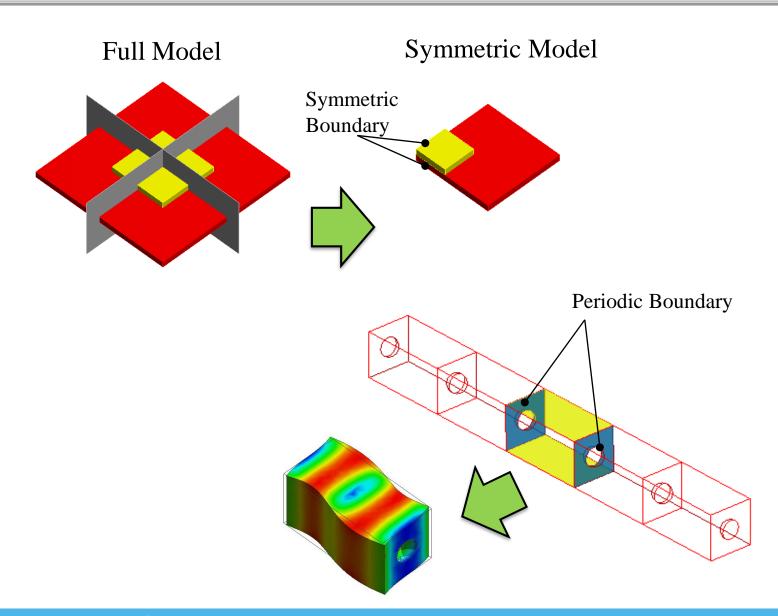
Water:  $Z = 997[kg/m3]*1500[m/s]=1.496e6[kg/(m2 \cdot s)]=1.4596e6[N \cdot s/m3]$ 





# 



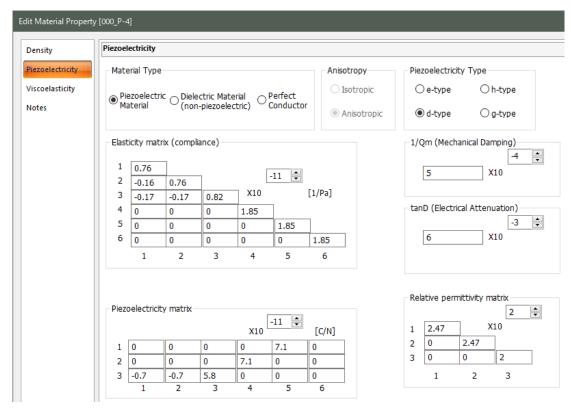


# (3) Material Property

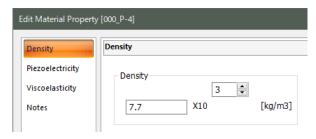


#### Piezoelectric Constants

Elasticity
Piezoelectricity
Permittivity
Loss (1/Qm, tanδ)

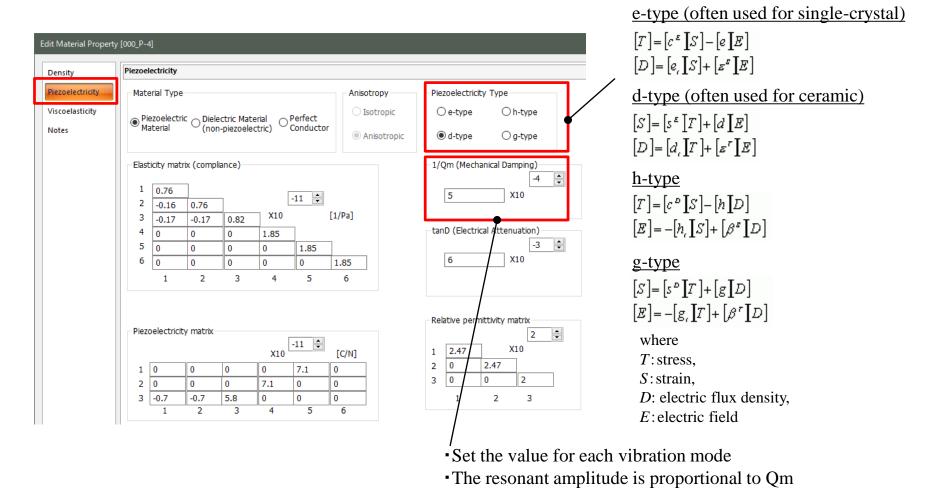


#### Density



### Piezoelectric Constant

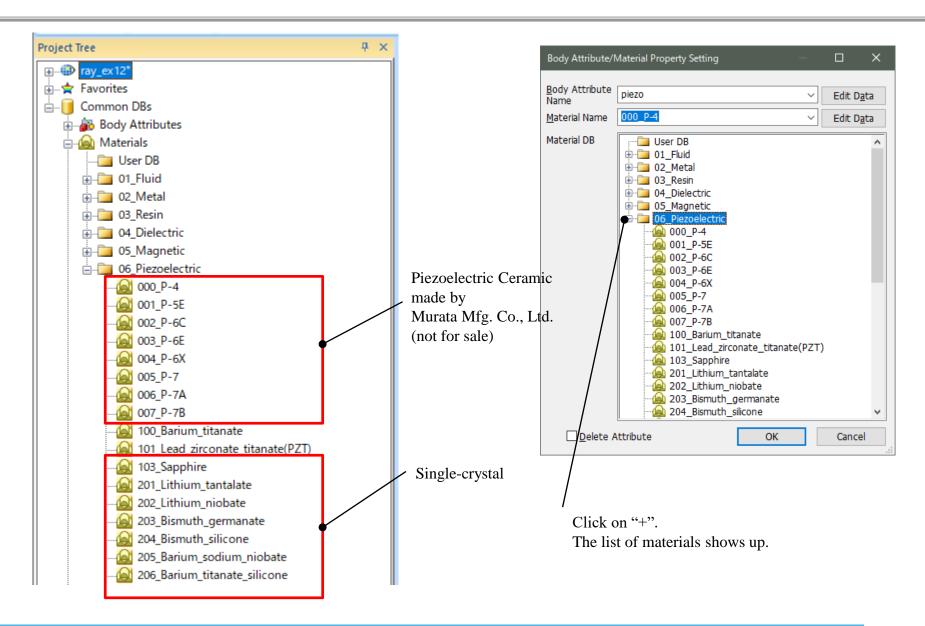




•Resonant resistance is inversely proportional to Qm

# Piezoelectric Materials in the Database Murata Software



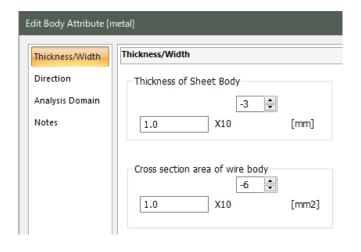


# (4) Body Attribute



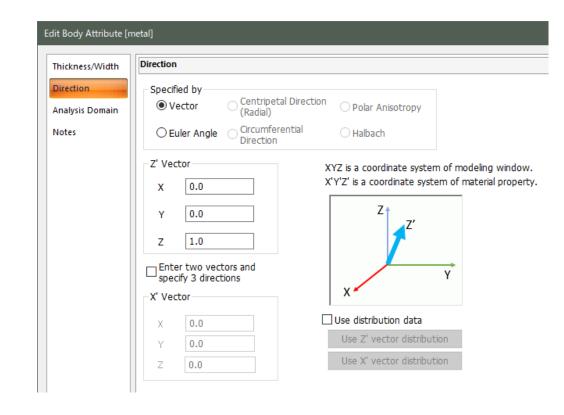
#### Thickness/Width

The thickness of sheet body is for the 3D analysis.



#### Direction

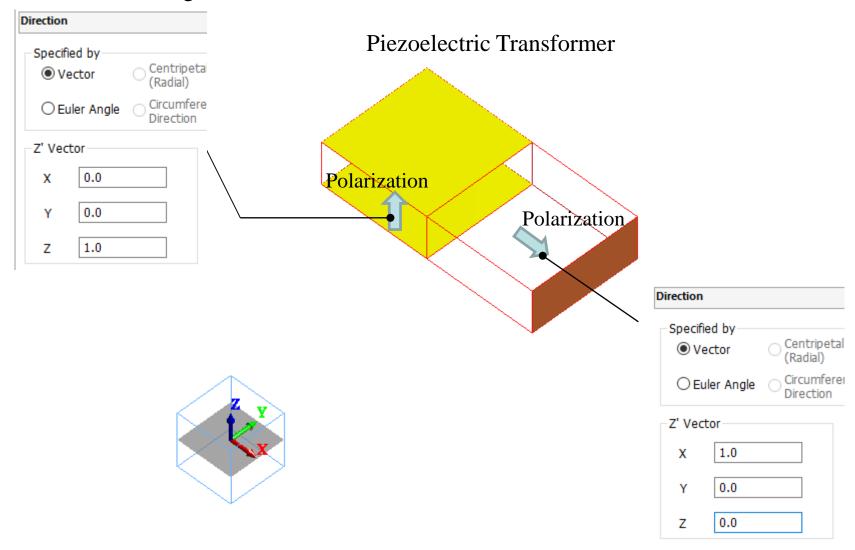
The axis direction of polarization and crystallization is specified.



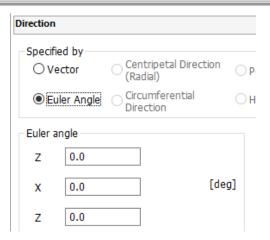
# Crystallization Axis Setting by Vector Murata Software



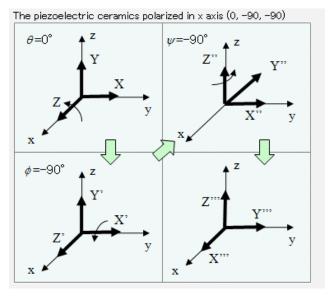
#### **Default Setting**

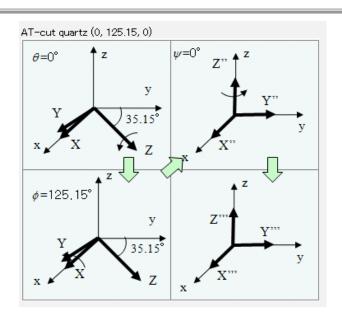


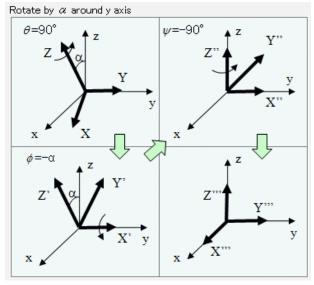
# Crystallization Axis Setting by Euler Angle Murata Software



Euler angle  $(\theta, \phi, \psi)$  is an angle rotated around Z axis by angle  $\theta$ , around X' axis by angle  $\varphi$ , and around Z" axis by angle  $\psi$ .





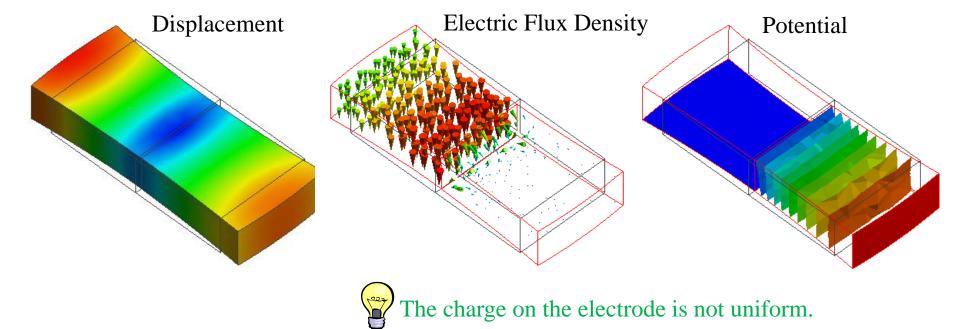


# (5) Results Display - Field



- Displacement [m]
- Strain
- Mechanical Stress [Pa]

- Potential [V]
- Electric Field [V/m]
- •Electric Flux Density [C/m<sup>2</sup>]



### Results Display

### Numerical Summary of Harmonic Analysis



- Charge of electrode with voltage specified [C]
- Current of electrode with voltage specified [A]
- Voltage of floating electrode [V]
- \*[Add a resistor across to ground] is deselected.

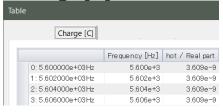


The sum of charges on the floating electrode is zero.

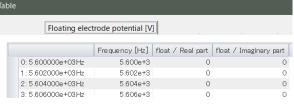
The current is zero as well.



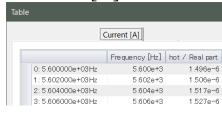
Tab "Charge | C



"Floating electrode potential [V]" Tab



"Current | A Tab



The data of the numerical table can be exported to csv file and used on Excel



### Results Display

### Numerical Summary of Resonant Analysis



Damping capacitance: Cd [pF]

•Free capacitance: Cf [pF]

•Resonant frequency: Fr[Hz]

• Difference between resonant freq. and anti-resonant freq.: DF[Hz]

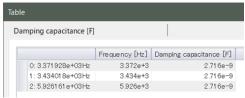
Coupling coefficient: k [%]

•Resonant resistance: Rn[ohm]

• Equivalent capacitance: Cn[pF]

• Equivalent inductance: Ln[H]

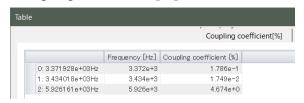
"Damping Capacitance [F]" Tab



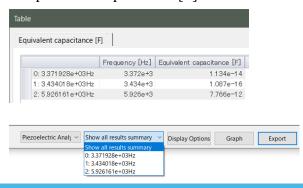
"Resonant Frequency [Hz]" Tab



"Coupling Coefficient [%]" Tab



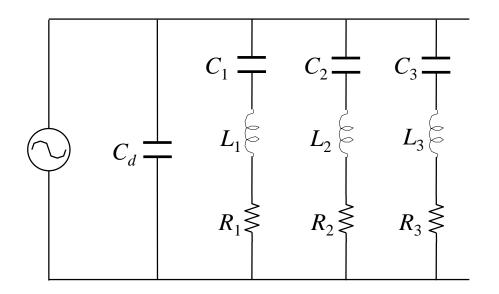
"Equivalent Capacitance [F]" Tab



# **Equivalent Circuit**



Equivalent circuit is obtained in the resonant analysis.



 $C_k$ : equivalent series capacitance

Lk: equivalent series inductance

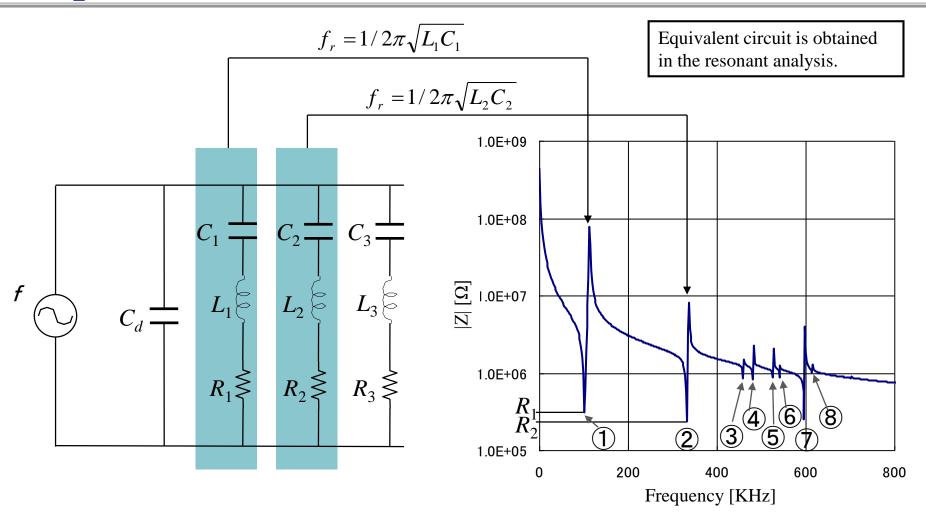
*Rk* : equivalent series resistance

*Cd* : damping capacitance

Cf: free capacitance

# Equivalent Circuit & Resonance Murata Software





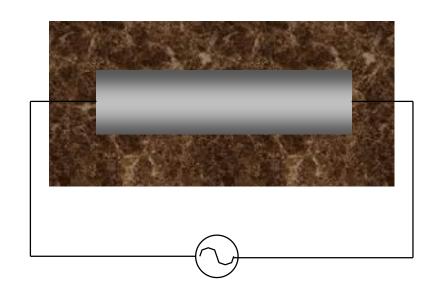
The equivalent circuit has multiple resonators. The number of resonators corresponds to the number of modes which is defined on the [Resonant Analysis] tab of the analysis condition setting. In the right figure above shows 8 resonant points, which means the calculation was performed with at least 8 modes.

The harmonic analysis results take into account all modes included in the range of analysis frequencies.

### Damping Capacitance & Free Capacitance

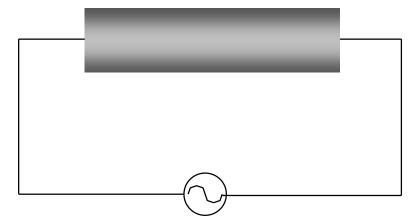
Equivalent circuit is obtained in the resonant analysis.

 $C_d$ : Damping Capacitance Cannot move freely



 $C_f$ : Free Capacitance Freely moving

$$C_f = C_d + \sum_k C_k$$



### Other Parameters



In the resonant analysis, resonant frequency, difference of the resonant frequency and anti-resonant frequency, and coupling coefficient are output.

#### **Resonant Frequency**

The resistance is at its minimum.  $f_r$ 

#### **Anti-resonant Frequency**

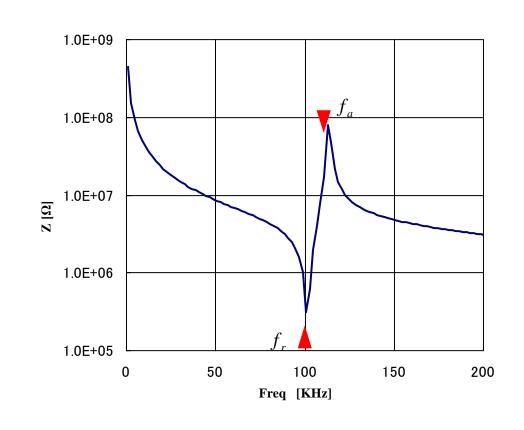
The resistance is at is maximum.

$$f_a = f_r \sqrt{1 + C_k / C_d}$$

#### **Coupling Coefficient**

Conversion efficiency between electrical and mechanical

$$k = \sqrt{C_k / C_f}$$



### Complex Resonant Frequency & Q

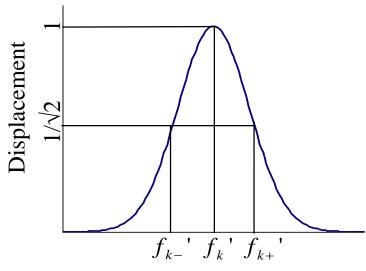


In the resonant analysis, complex resonant frequency is obtained by entering Q of material,  $tan\delta$ , and acoustic impedance. Frequency response of displacement is obtained in the harmonic analysis.

Q is the ratio of the real part and the imaginary part.

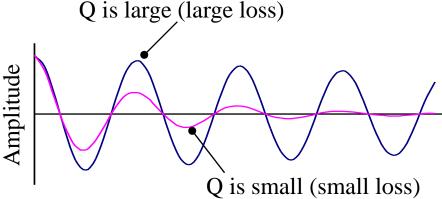
$$f_{k} = f_{k}' + jf_{k}''$$

$$Q = \frac{f_{k}'}{f_{k+}' - f_{k-}'} = \frac{1}{R_{k}} \sqrt{\frac{L_{k}}{C_{k}}} = \frac{f_{k}'}{2f_{k}''}$$



Frequency





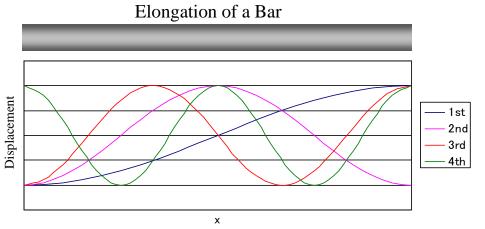
# 3. Points of Piezoelectric Analysis Murata Software



- (1) Mesh Size
- (2) Symmetric Model

### (1) Mesh Size



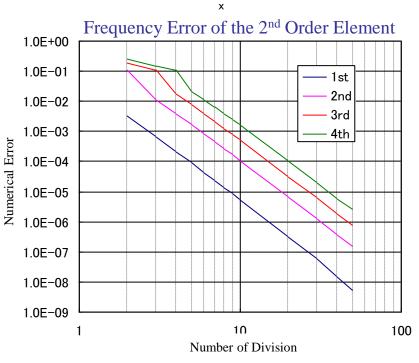


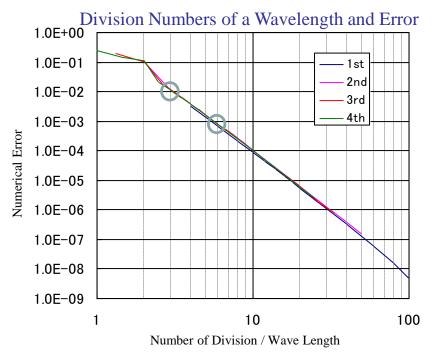
Frequency of the vertical vibration

$$f_n = \frac{n}{2L} \sqrt{\frac{E}{\rho}}$$
  $(n = 1,2,3,4)$ 



3 divisions of a wavelength: 1% 6 divisions of a wavelength: 0.1%

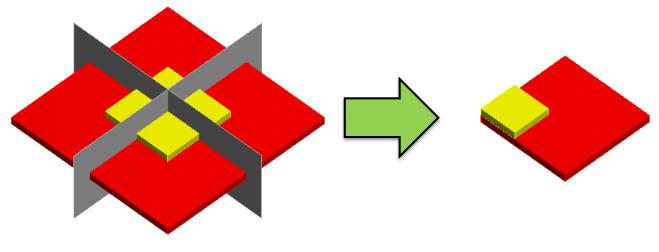




# (2) Symmetric Model



By analyzing a part of the domain, calculation is performed in shorter time with less memory usage.



Full Model

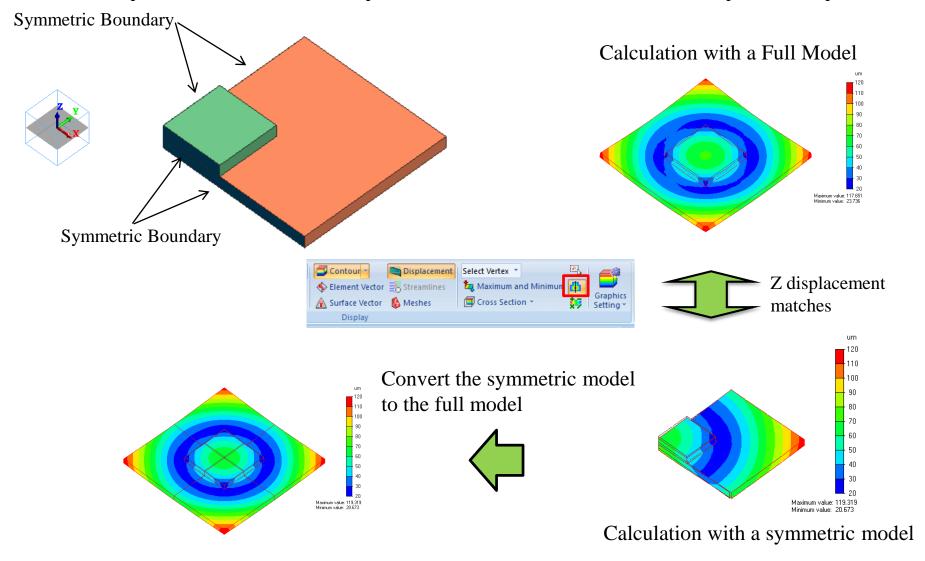
Symmetric Model

Domain	Memory	Time
1/2	1/2	1/4 (25.0%)
1/4	1/4	1/16 ( 6.3%)
1/8	1/8	1/64 ( 1.6%)

# (2) Symmetric Model



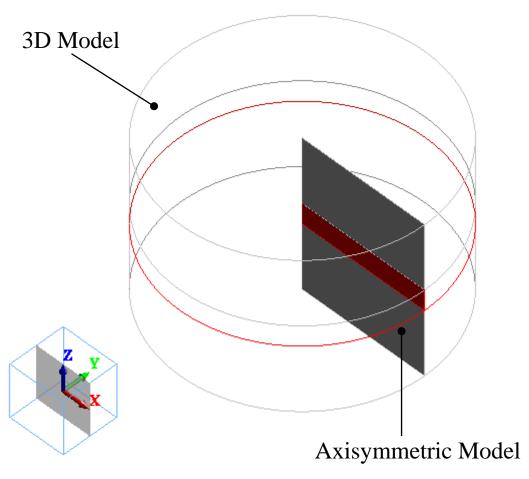
Set symmetric boundary condition on the faces of symmetry.

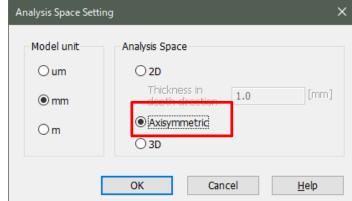


# (2) Symmetric Model



In the axisymmetric analysis of XZ plane, the analysis time is greatly reduced.





### Table of Contents



#### **☆**Piezoelectric Analysis

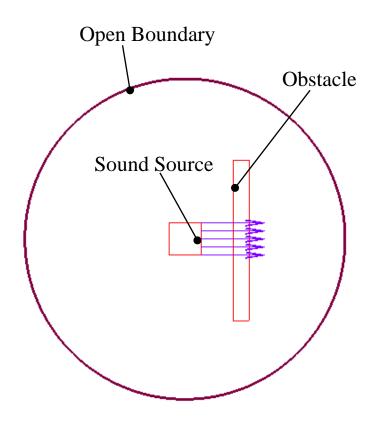
- 1. Case Studies
- 2. Functions
- 3. Points

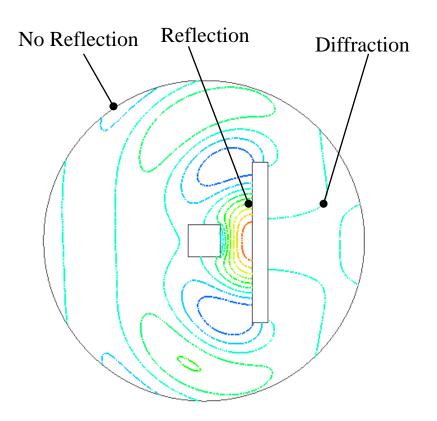
#### **☆**Acoustic Analysis

- 4. Case Studies
- 5. Functions
- 6. Points
- **☆**Piezoelectric-Acoustic Coupled Analysis
- 7. Case Studies
- 8. Points



#### Reflection and Diffraction

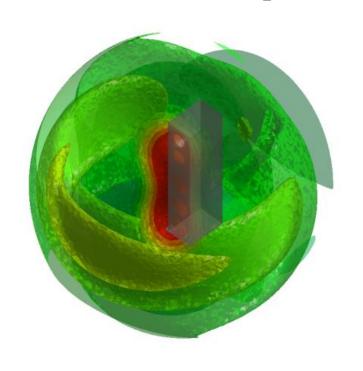




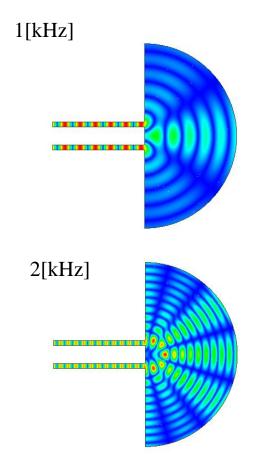
# 



Sound Pressure of Speaker



#### Interference of Sound Waves



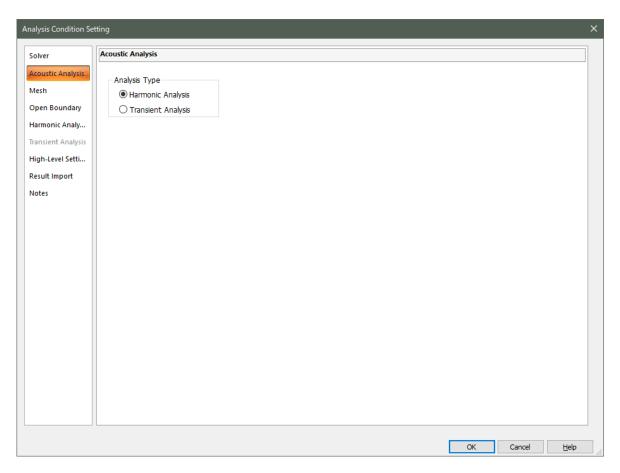
### 5. Functions of Acoustic Analysis



- (1) Analysis Condition
- (2) Boundary Condition
- (3) Material Property
- (4) Body Attribute
- (5) Results Display

# (1) Analysis Condition

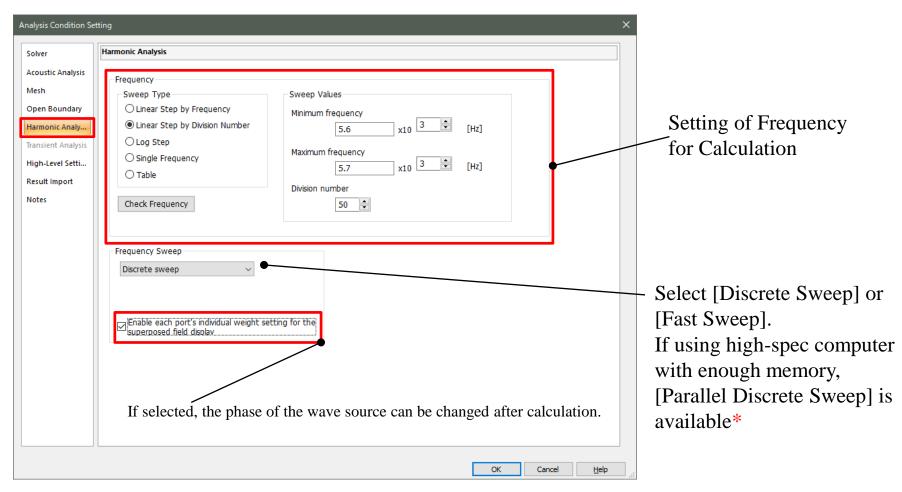




Analysis Type
Harmonic
Transient

## Harmonic Analysis

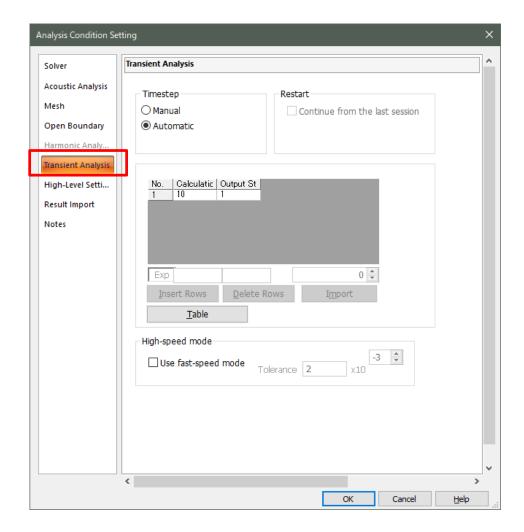




\*Option for Accelerator is required.

## Transient Analysis



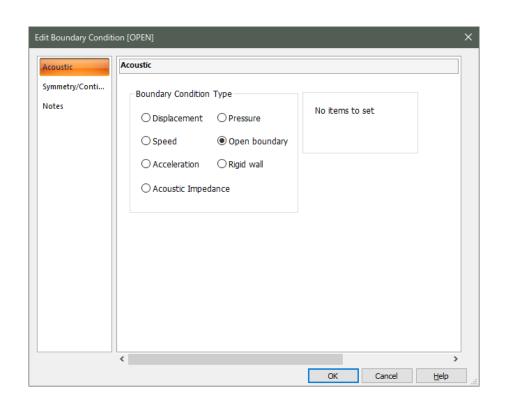


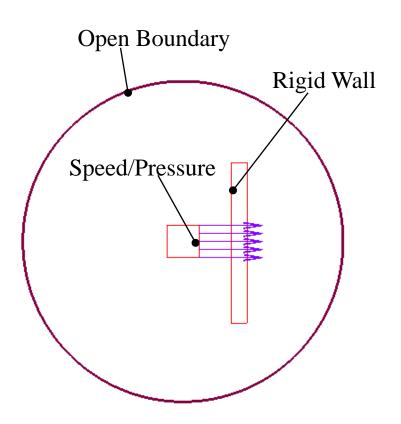
In the acoustic transient analysis, if timestep is set [Automatic], timestep is set based on the CFL number so as to prevent numerical instability.

\*CFL Number is the number of meshes that travel during one timestep. If the number is too large, numerical instability occurs. The number is determined by the mesh size and the sound speed.

# (2) Boundary Condition



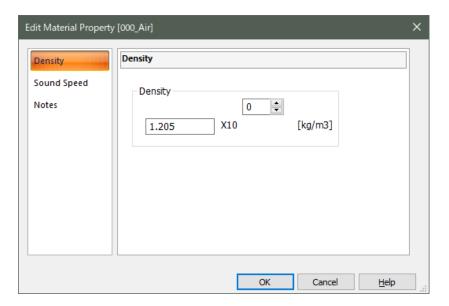




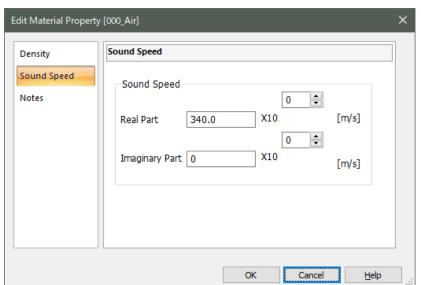
# (3) Material Property



Density



Sound Speed



# Damping of Sound



By setting imaginary part of the sound speed, the damping of the sound waves can be expressed.

Fig 1 shows the damping material between air.

As Fig 2 shows, the sound waves are damped in the damping material.

Fig 3 shows the level on the left side of the damping material is not uniform. It indicates the reflection is occurring.

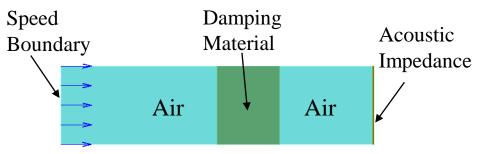


Fig 1. Sound Pressure [Pa]

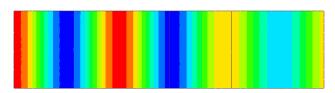


Fig 2. Sound Pressure [Pa]

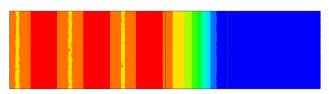


Fig 3. Sound Pressure Level [dB]

## Damping of Sound



#### Loss of the Medium

	Sound Speed													
_	ound Speed													
	Sound Speed													
	Sound Speed		0											
	Real Part	340.0	X10	[m/s]										
			0											
	Imaginary Part	5	X10	[m/s]										

Set the imaginary part and damping coefficient of the sound speed here.

1 The imaginary part indicates the attenuation, which sometimes cannot be ignored in the case of ultrasonics. Kirchhoff's equation is given below. The sound speed increases as the frequency increases.

$$c_{image} = \frac{\omega}{2c} \left( \frac{4}{3} \frac{\mu}{\rho} + \frac{\gamma - 1}{\gamma} \frac{\kappa}{\rho C_{\nu}} \right)$$

 $\omega$  : angular frequency, c : sound speed,  $\mu$  : viscosity coefficient,  $\rho$  : density

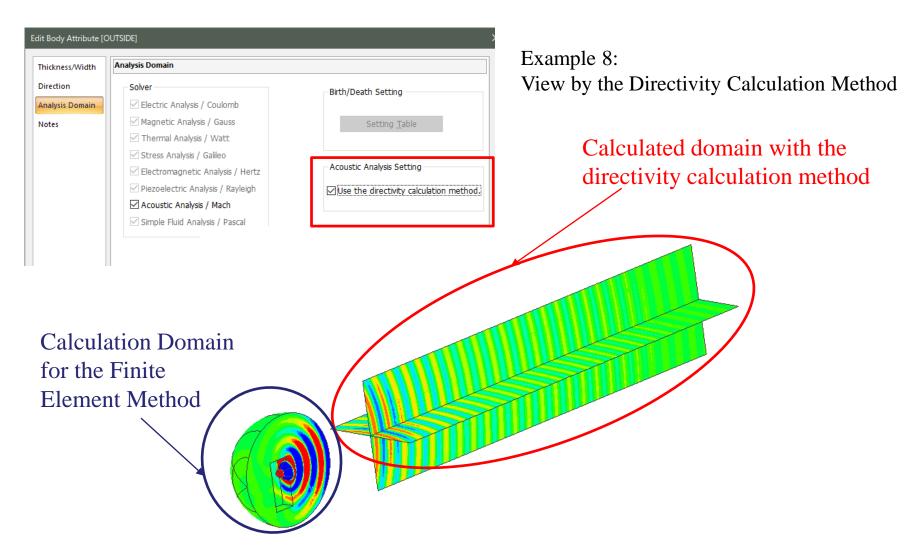
 $\gamma$  : heat capacity ratio,  $\kappa$  : thermal conductivity,  $\operatorname{Cv}$  : specific heat at constant volume

It is  $2.7 \times 10^{-7} f$  in the air, and  $2.8 \times 10^{-9} f$  in the water, where f is the frequency.

# (4) Body Attribute

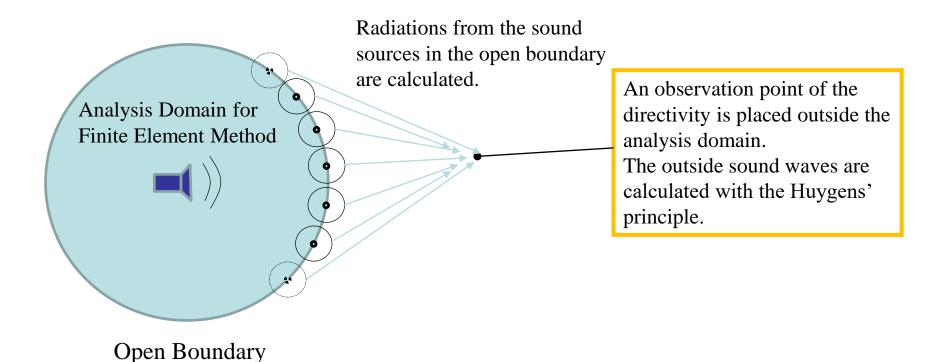


### **Analysis Domain**



# Calculation Method of Directivity Murata Software

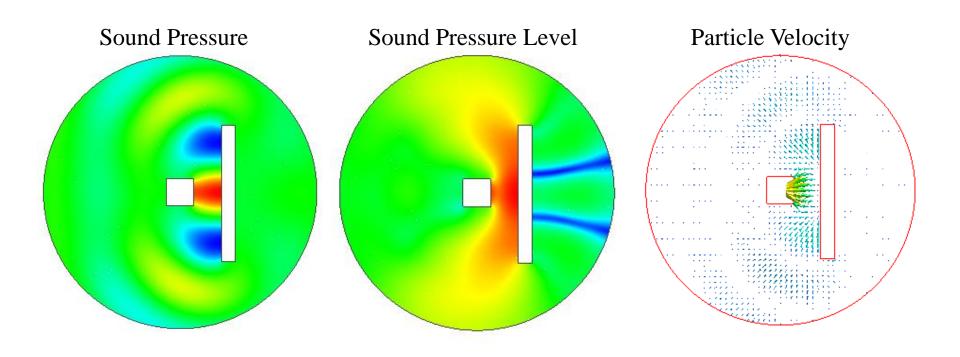
### Huygens' Principle



# (5) Results Display - Field



Sound Pressure [Pa]
Sound Pressure Level[dB]
Particle Velocity [m/sec]
Acoustic Intensity [W/m²]



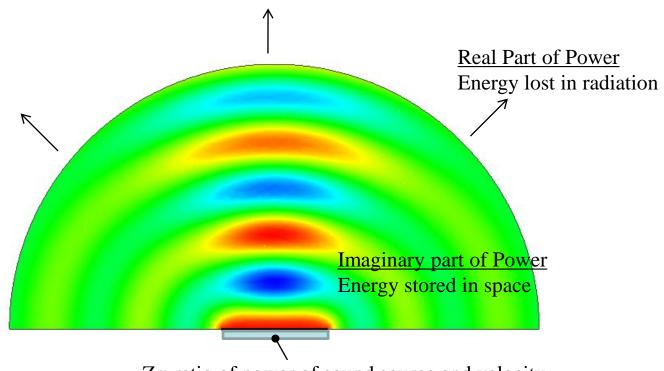
### Results Display - Numerical Table



Radiation Energy: Power[W]
Radiation Impedance: Zr[Ns/m]

#### Example

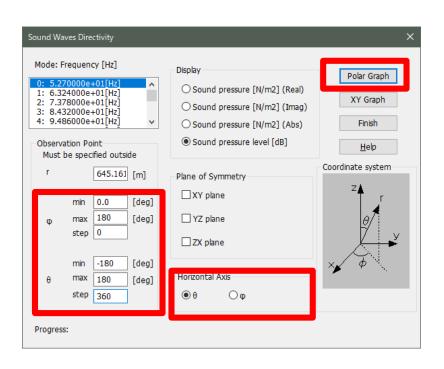
Power [W] = 2.60245624e-005 7.47853134e-005j Zr [Ns/m] = 4.33208345e+000 1.24488632e+001j



Zr: ratio of power of sound source and velocity

### Results Display - Graph of Directivity





Based on the Huygens' principle, the sound pressure outside the domain is calculated, where the finite element method is not applied.

#### [XY Plane]

Φ:min 0/max 360/step 100

 $\theta$ : min 0/max 0/step 0

Horizontal axis: Φ

#### [XZ Plane]

 $\Phi$ : min 0/max 0/step 0

Θ:min -180/max 180/step 100

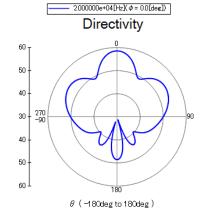
Horizontal axis:  $\theta$ 

#### [YZ Plane]

Φ: min 90/max 90/step 0

Θ:min -180/max 180/step 100

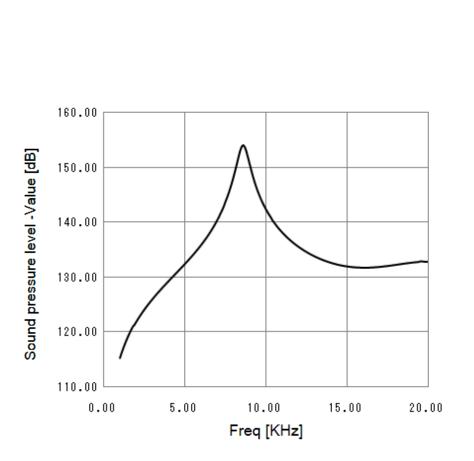
Horizontal axis:  $\theta$ 

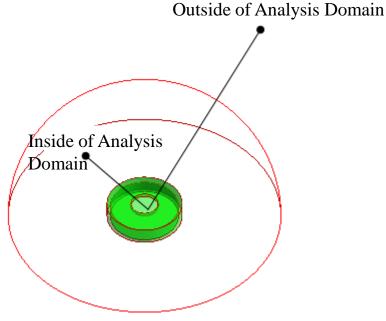


Example of Directivity of [XZ Plane]



#### Frequency Response of Sound Pressure





Inside of Analysis Domain

Use Graph function of the contour diagram.

Outside of Analysis Domain
Refer to [Acoustic Analysis Example 8].
Or use macro in the Technical Note [Frequency Response of Sound Pressure Level].

# 6. Points of Acoustic Analysis

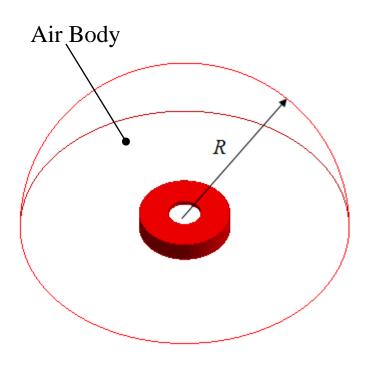


- (1) Analysis Domain
- (2) Mesh Size
- (3) Multiple Sound Sources
- (4) Frequency-dependent Sound Speed

# (1) Analysis Domain



#### Create air body around the sound source



The radius of the air body is R.

R is 0.2 times the wavelength of the frequency of your interest.

If the value is small, a warning will show up as below.

\*\* Warning \*\*

Distance between the wave source and the open boundary is too short. Separate them at least 0.2 wavelength.

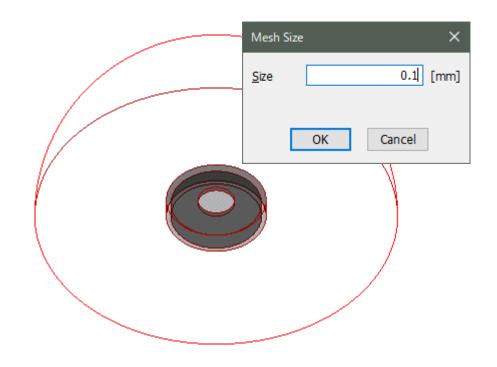
# (2) Mesh Size



Mesh size is somewhere between a fourth and sixth of the wavelength.

Project Tree Д X Ė-₩ Ex2\* Model Model unit: m/3D Analysis Condition: Acoustic/Harmonic Analysis Body Attributes Materials Boundary Conditions Mesh Sizes Mesh Size : 0.5 ∴ X Variables i... 👍 Results\* 🖮 🍞 Field 🚣 Global Coordinates Table ☐ M Chart Common DBs

Finer mesh must be applied to the complicated form near the sound source.



# (2) Mesh Size



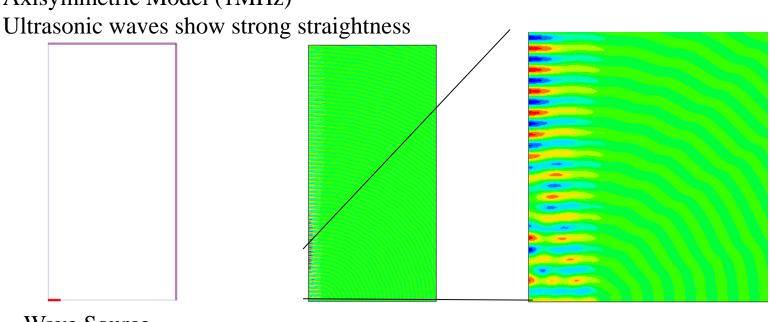
#### Ultrasonic Analysis

To prepare the mesh of a sixth of the wavelength, the number of meshes becomes large if the analysis domain is wide.

In such case, a quarter model or axisymmetric model are helpful.

#### [Example]

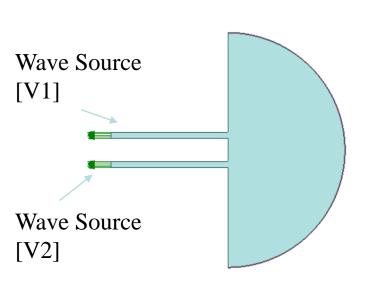
Axisymmetric Model (1MHz)

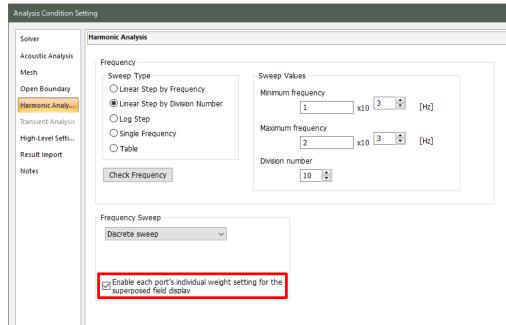


Wave Source

# (3) Multiple Sound Sources





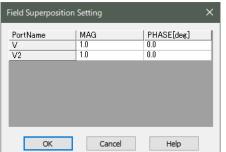


Select [Enable each port's individual weight setting for the superposed field display]

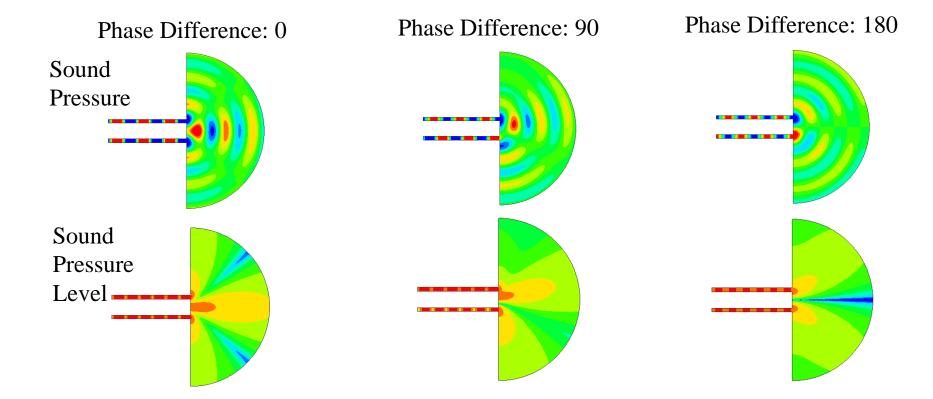
Acoustic Analysis Example 3

# (3) Multiple Sound Sources





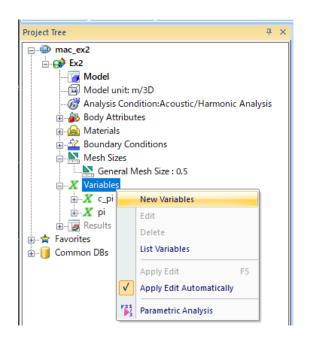
Scale factor and phase can be shifted for each boundary condition of the results.



### (4) Frequency-dependent Sound Speed

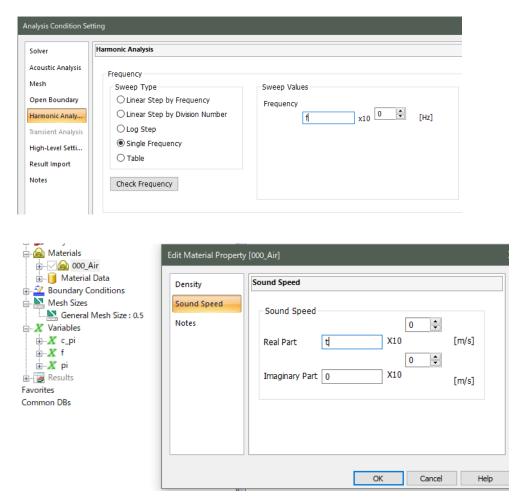


#### Project Tree > Right-click on Variables > Select [New Variables]





Prepare variables f and t. Initial value can be any values.

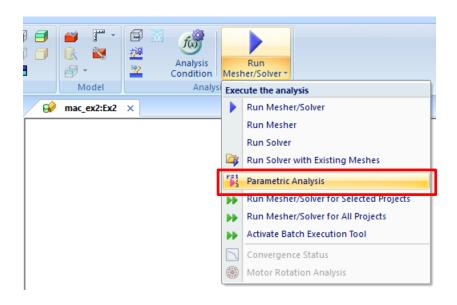


Analysis Condition Setting > Harmonic Analysis > Set f to Frequency. Edit Material Property > Set t to Sound Speed.

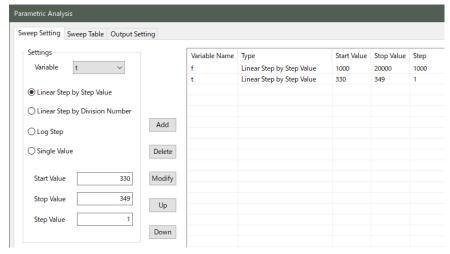
### (4) Frequency-dependent Sound Speed



#### Run Mesher/Solver > Submenu > Parametric Analysis



By setting in the [Sweep Table], analysis can be performed while changing frequencies (f) and sound speed (t).



arametric Ana	lysis				
Sweep Setting	Sweep Table	Output Setting			
Variable	f	t			
1	1000.0	330.0			
2	1000.0	331.0			
3	1000.0	332.0			
4	1000.0	333.0			
5	1000.0	334.0			
6	1000.0	335.0			
7	1000.0	336.0			
	1000.0	337.0			
9	1000.0	338.0			
	1000.0	339.0			
11	1000.0	340.0			
	1000.0	341.0			
13	1000.0	342.0			
	1000.0	343.0			
	1000.0	344.0			
	1000.0	345.0			
	1000.0	346.0			
	1000.0	347.0			
	1000.0	348.0			
	1000.0	349.0			
	2000.0	330.0			
	2000.0	331.0			
0.0	2000 0	222			

### Table of Contents



#### **☆**Piezoelectric Analysis

- 1. Case Studies
- 2. Functions
- 3. Points

#### **☆**Acoustic Analysis

- 4. Case Studies
- 5. Functions
- 6. Points

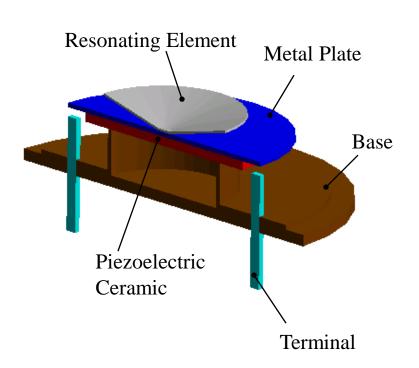
### **☆**Piezoelectric-Acoustic Coupled Analysis

- 7. Case Studies
- 8. Points

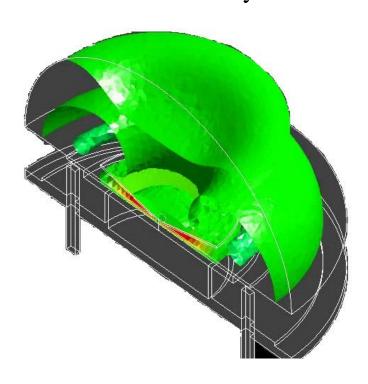


#### Ultrasonic Sensor

#### Piezoelectric Analysis



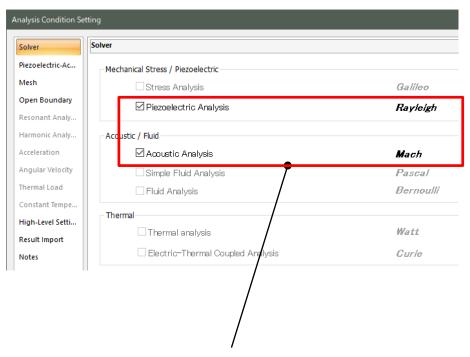
#### **Acoustic Analysis**



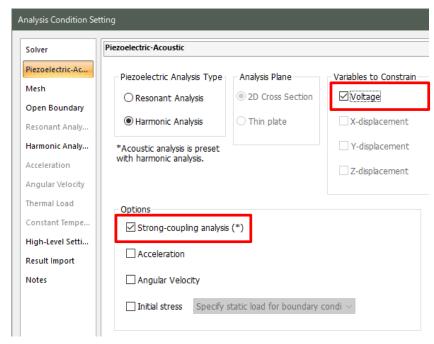
# 8. Points of Piezoelectric-Acoustic Coupled Analysis



### **Analysis Condition**



Select [Piezoelectric Analysis] and [Acoustic Analysis]

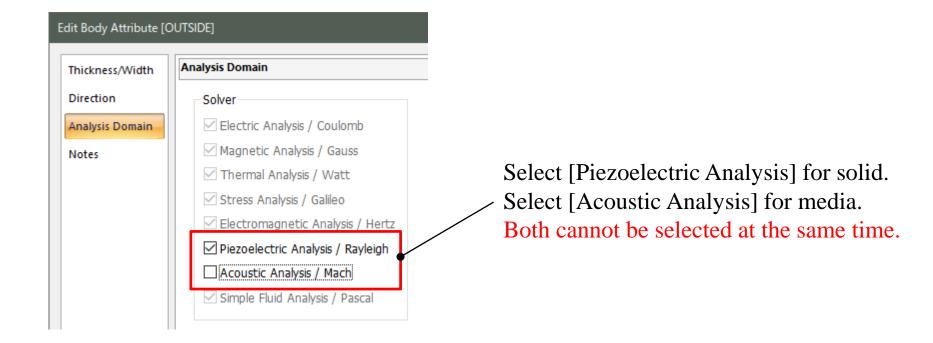


Select [Strong-coupling analysis]. Select [Voltage] if [piezoelectricity] is not taken into account.

# 8. Points of Piezoelectric-Acoustic Coupled Analysis



### **Body Attribute**



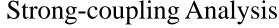
# Strong-coupling Analysis



#### Weak-coupling Analysis

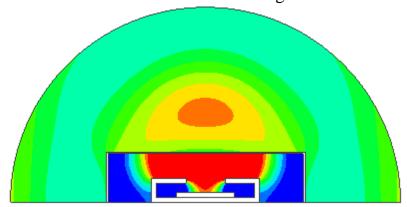
(Unidirectional from piezoelectric to acoustic)

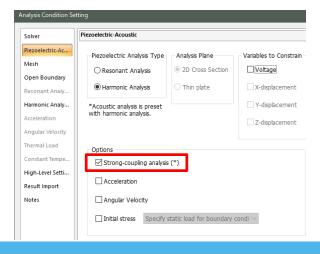
The vibrating effect of the structure by the sound waves cannot be calculated. The sound waves outside the shielding plate cannot be calculated either.

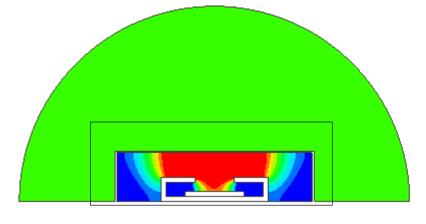


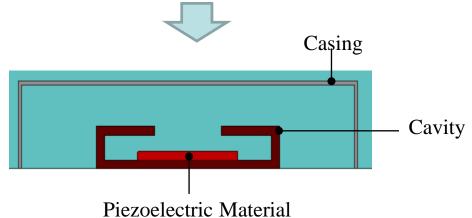
(Bidirectional between piezoelectric and acoustic)

The vibrating effect of the structure by the sound waves can be calculated and the sound waves outside the shielding plate can be calculated as well. However the calculation time is long.









# Thank You