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Q: Can Femtet analyze the vibration test?

A: Yes, Femtet can do it with the harmonic analysis of the stress solver.

You can observe how the vibration pattern will change in response to the frequency of the forced vibration in the analysis result (See Example 50 of the stress analysis).

Performing the resonant analysis and harmonic analysis sequentially allows you to efficiently execute the vibration analysis while focusing on the resonant frequency. See the detailed material for specific procedures.



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Vibration Test Analysis





- Modeling
- Resonant analysis to obtain resonant frequency
- Harmonic analysis to simulate the vibration test

Note

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- The displacement and stress in the resonant analysis are relative values (Absolute values cannot be acquired).
- The displacement and stress in the harmonic analysis are absolute values (Mechanical loss must be set).
- If a resonant frequency is within the range of vibration test frequencies, the displacement and stress at the resonant frequency will be so large. Then sweeping the frequency by fine steps around the resonant frequency is required to catch the peaks of displacement and stress.
- When the frequency range A of the vibration test does not entirely include the frequency range B that covers the resonant frequencies acquired through the resonant analysis;

if B < A, change the number of modes in the resonant analysis and then perform the resonant analysis, and

if A < B, perform the harmonic analysis with frequency properly swept in the range of A without additional resonant analysis.



Execute the harmonic analysis for the resonant modes of which frequencies are within the frequency range of the vibration test. The diagram above represents that three resonant modes are selected.

Procedure after Resonant Analysis (2)



If the frequencies of the resonant modes are out of the frequency range for the vibration test, perform the harmonic analysis with frequencies properly swept in

the vibration test range.



If the frequencies of the resonant modes obtained from the resonant analysis are lower than the frequency range for the vibration test, increase the number of modes for analysis.

Consequently, any resonant modes of which resonant frequencies are within the frequency range for the vibration test may be obtained.

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Boundary Condition (Vibration Setting)

Mechanical	Mechanical					
Symmetry/Conti Notes	Boundary Conditio	n Type Lumped Vertex Load Distributed Edge Load Distributed Face Load Orraye Load Olonit Load m/s2	Simple Contact Surface Spring Connection Remote Load	Acoustic Impedance Open Open Free	Time Dependency Weight Function Unform Duspacement UX UX	Set ON/OFF ON/OFF Lst Use distribution data Distribution Data Results Import
	Z 0.0	ormal displacement			Aj to vi -S	pply t the p bratic et disp

Apply the following boundary conditions to the portion, or face, mounted on the vibration test machine,

-Set displacement if an amplitude vibration, -Set acceleration if an acceleration vibration.

To fix displacements in all directions except for the acceleration direction, select [Constrain the normal displacements].

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Frequency Sweep in Harmonic Analysis



The displacement and stress become maximum at the resonant frequency. Sweep the frequency by such a resolution that the peak of the displacement and stress can be caught.

The sharpness of the peak depends on tan δ , or 1/Qm.

Mechanical Loss



Edit Material Propert	y [002_Polycarbonate(PC)]			×
Density	Elasticity			
Elasticity Creep Viscoelasticity Hyperelasticity Notes	Material Type © Elastic/Isotropic ○ Elastic/Anisotropic ○ Elasto-plastic/Bilinear ○ Elasto-plastic/Multilinear Young's modulus 2.24 Poisson's ratio 0.3	Temperature Dependency No Yes Elasticity Matrix Type Stiffness Compliance	Hardening Law I Sotropic Hardening Kinetic Hardening LanD (Mechanical Damping) O.0	
			OK Cancel <u>H</u> elp	

If mechanical loss is zero, the displacement and stress at the resonance peak will be infinite. If mechanical loss, or tan δ , is set, damping occurs to suppress the displacement and stress at the resonance peak to finite values.



The case study below represents the following steps: Obtain the resonant frequencies through the resonant analysis, and Perform the harmonic analysis at the resonant frequencies.

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The model is a quarter-symmetric model of a substrate with a component mounted at the center.

The symmetric face is set with a symmetric boundary condition (Fix). Fixed displacement boundary condition is set to the inner surface of the hole located at the corner of the substrate.

Analysis Condition (Harmonic Analysis)

Analysis Condition Setting



The number of precalculated modes can be changed at the number of modes on the [Resonant Analysis] tab. The default is 3.

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The resonant analysis indicates the fundamental frequency is 354.4579 Hz. The displacement diagram shows the shape of a vibration mode. (The displacements are relative values.)

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The model is a quarter-symmetric model of a substrate with a component mounted at the center.

The symmetric face is set with a symmetric boundary condition (Fix).

The acceleration boundary is set to the inner surface of the hole located at the corner of the substrate.

See the next slides for more information.

Boundary Condition (Acceleration)



Edit Boundary Condition [Boundary_Condition_001]

Mechanical	Mechanical				
Symmetry/Conti Notes	Boundary Condition	D Type Lumped Vertex Load Distributed Edge Load	O Simple Contact Contact Surface	Acoustic Impedance Open Boundary	
	Displacement	O Distributed Face Load	○ Spring Connection	OFree	
		O Pressure	ORemote Load		-
) Joint Load			
	X 0.0				(
	Y 0.0	m/s2 ~			
	Z 10*9.8				,
	✓ Constrain the n	ormal displacement			
					- (

A vibrational acceleration of 10 G is applied in the Z direction.

To fix displacements in all directions except for the acceleration direction, select [Constrain the normal displacements]. The diagram on the left indicates the displacements in the XY direction are constrained. *

* It represents the state where the substrate is fixed to the test fixture.

Analysis Condition (Harmonic Analysis)

Analysis Condition Setting Stress Analysis Solver Harmonic Analysis Solver Stress Analysis Analysis Type Stress Analysis Frequency Mesh Mesh ○ Static Analysis Sweep Values Sweep Type Resonant Analy. Resonant Analy... Linear Step by Frequency O Resonant Analysis Minimum frequency Harmonic Analy... CLinear Step by Division Number Harmonic Analy... 352 Harmonic Analysis Hz \sim Transient Analysis ○Log Step Transient Analysis OBuckling Analysis Step/Thermal Lo... Maximum frequency ○Single Frequency Step/Thermal Lo... 358 Acceleration O Transient Analysis Hz \sim ○ Table Acceleration Angular Velocity Frequency step Options Angular Velocity Constant Tempe... Check Frequency 0.05 Hz Acceleration High-Level Setti... Constant Tempe.. Results Import Angular Velocity High-Level Setti... Notes Results Import Constrain the freedom of s Notes Calculate the mass and the

The sweep values are set such that the frequency of the peak for the resonant mode around 355 Hz falls within the frequency sweep range.

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Mechanical Loss Setting



Elasticity	Elasticity			
Material Type	Temperature Dependency	Hardening Law		
sticity e) Elastic/Isotropic e) Elastic/Anisotropic Elasto-plastic/Bilinear Elasto-plastic/Multilinear	No Yes Institute Stiffness Compliance	Isotropic Hardening Kinetic Hardening tanD (Mechanical Damping)		
Young's Modulus				
Poisson's Ratio				

Calculating the absolute values of displacement and stress at the resonance peak requires setting mechanical loss.

The diagram above indicates the mechanical loss of the substrate of glass epoxy is set to 0.001.

Result (Harmonic Analysis)

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Maximum displacement / X Component Maximum displacement / Y Component Maximum displacement / Z Component

Maximum displacement



The graph is based on the maximum displacement listed in the numerical summary table. It indicates a peak of maximum

displacement at nearly the same frequency as the resultant frequency obtained from the resonant analysis.

Result (Harmonic Analysis)

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The diagrams show the maximum magnitudes of maximum principal stress and displacement at the resonance peak. As the values of each field vary depending on the phase setting, select [Maximum] in the phase to display the maximum values.